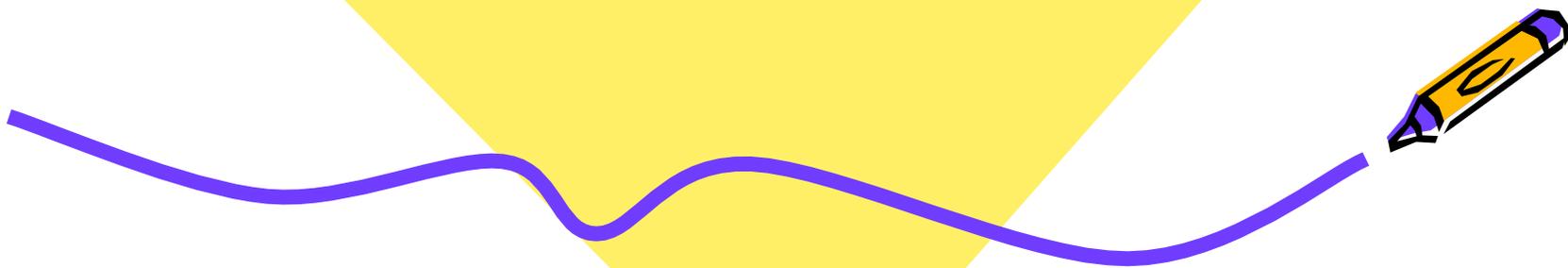


第十章

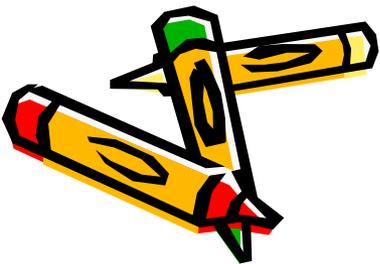
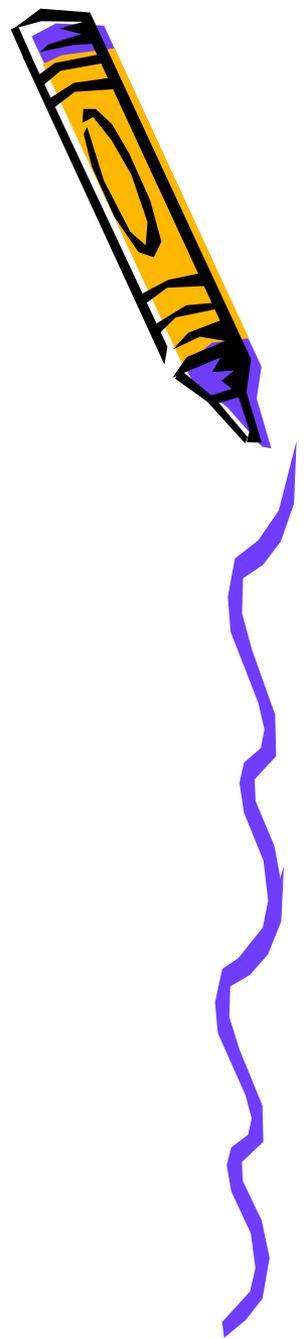
传热过程分析 与换热器热计算



§ 10-1 传热过程的分析 and 计算

传热过程： 高温流体通过固体壁面把热量传给低温流体的过程。

传热量计算： $\Phi = kA(t_{f1} - t_{f2})$



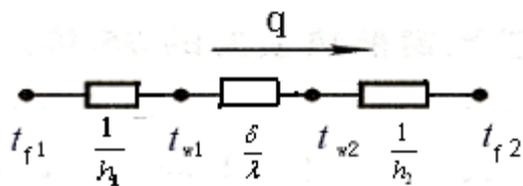
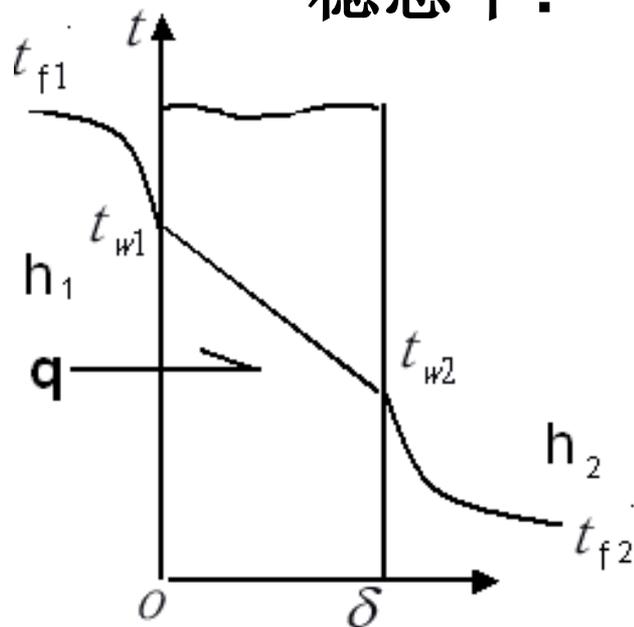
一、传热系数的计算

1. 通过平壁的传热

稳态下:

$$\Phi = \frac{(t_{f1} - t_{f2})A}{\frac{1}{h_1} + \frac{\delta}{\lambda} + \frac{1}{h_2}} = kA(t_{f1} - t_{f2}) \quad (W)$$

$$\therefore k = \frac{1}{\frac{1}{h_1} + \frac{\delta}{\lambda} + \frac{1}{h_2}} \quad (W / m^2 \cdot K)$$

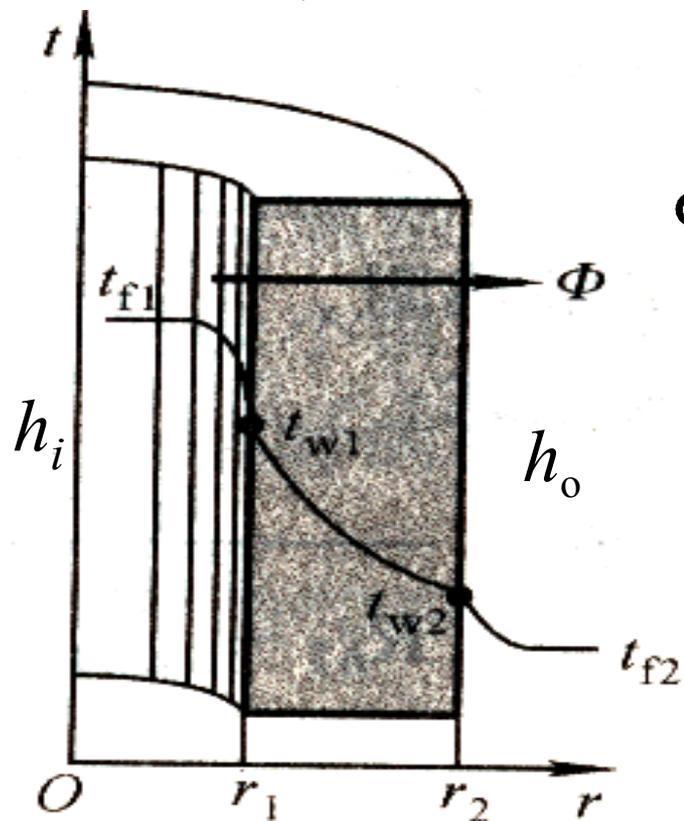


A两侧相同, $\therefore k$ 也两侧相同

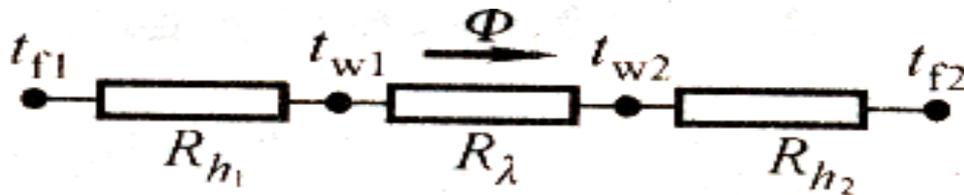


2. 通过圆管的传热

稳态下，总热流量是不变的。



$$\Phi = \frac{t_{f1} - t_{f2}}{\frac{1}{h_i \pi d_i l} + \frac{1}{2\pi\lambda l} \ln \frac{d_o}{d_i} + \frac{1}{h_o \pi d_o l}}$$
$$= k_0 \pi d_o l (t_{f1} - t_{f2}) \quad (W)$$

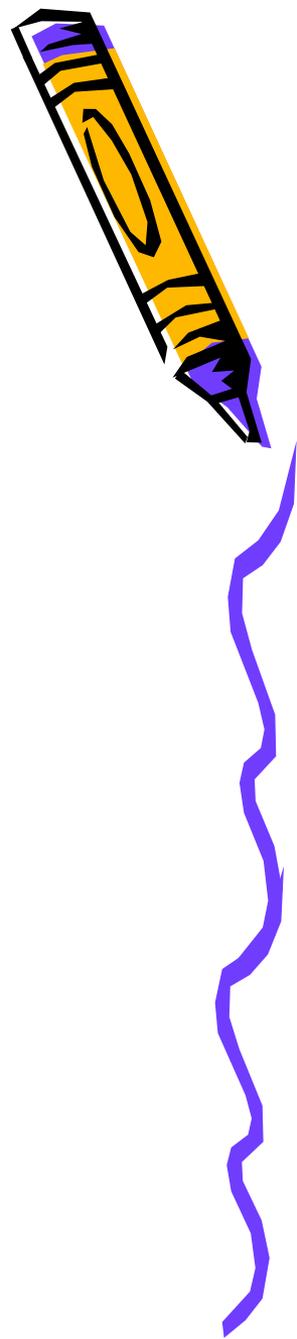
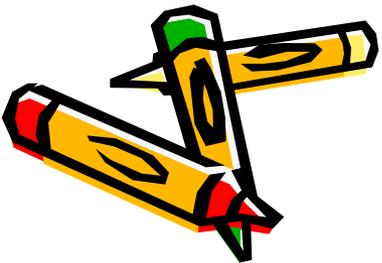


$$\Phi = k_0 \pi d_0 l (t_{f1} - t_{f2}) \quad (W)$$

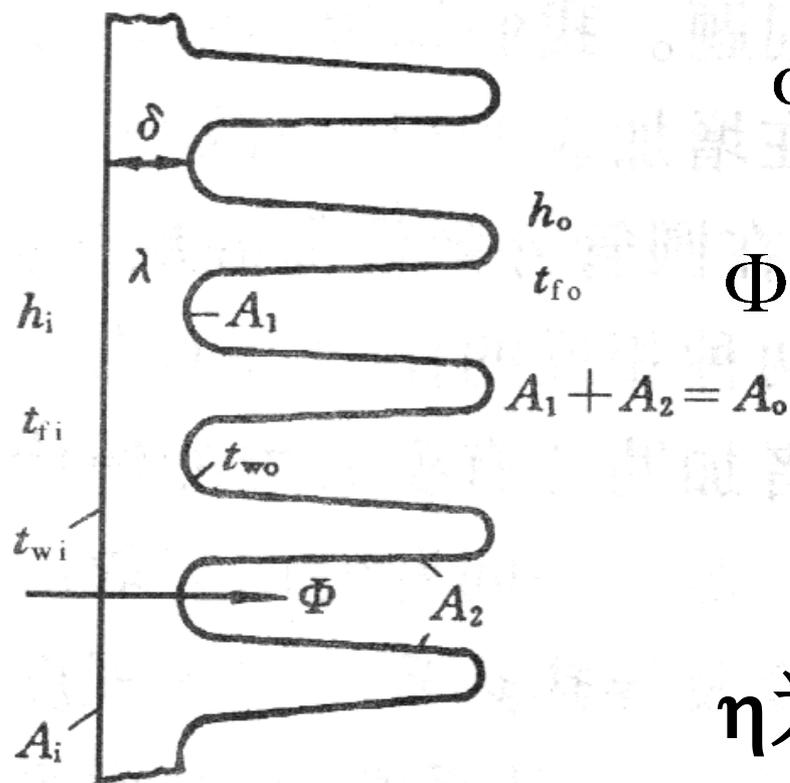
$$k_0 = \frac{1}{\frac{1}{h_i} \cdot \frac{d_o}{d_i} + \frac{d_o}{2\lambda} \ln \frac{d_o}{d_i} + \frac{1}{h_o}} \quad (W / m^2 \cdot K)$$

$$\Phi = k_i \pi d_i l (t_{f1} - t_{f2}) \quad (W)$$

$$k_i = \frac{1}{\frac{1}{h_i} + \frac{d_i}{2\lambda} \ln \frac{d_o}{d_i} + \frac{1}{h_o} \frac{d_i}{d_o}} \quad (W / m^2 \cdot K)$$



3. 通过肋壁的传热



$$\Phi = h_i A_i (t_{fi} - t_{wi})$$

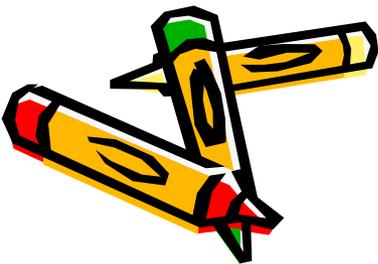
$$\Phi = \frac{\lambda}{\delta} A_i (t_{wi} - t_{wo})$$

$$\Phi = h_o A_1 (t_{wo} - t_{fo}) + h_o \eta_f A_2 (t_{wo} - t_{fo})$$

$$= h_o \eta A_0 (t_{wo} - t_{fo})$$

η 为肋面总效率:

$$\eta = \frac{A_1 + A_2 \eta_f}{A_0}$$



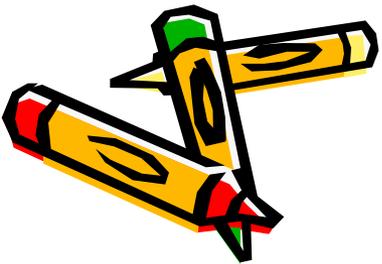
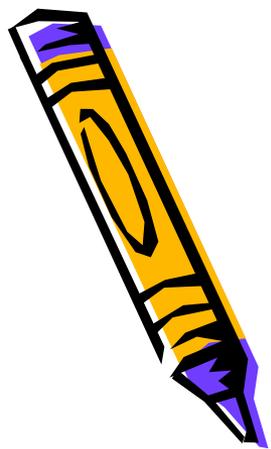
$$\Phi = \frac{t_{f1} - t_{f2}}{\frac{1}{h_i A_i} + \frac{\delta}{\lambda A_i} + \frac{1}{h_o A_o \eta}}$$

则以光壁为基准的 传热系数:

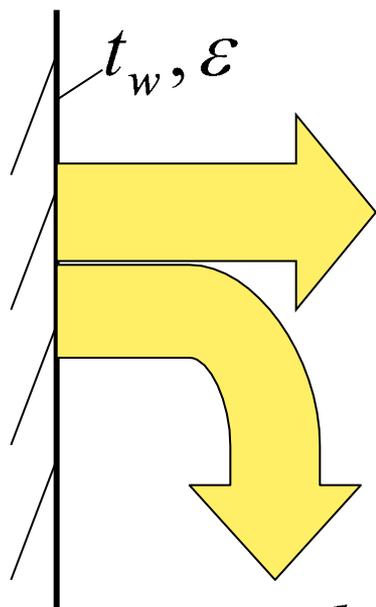
$$k_i = \frac{1}{\frac{1}{h_i} + \frac{\delta}{\lambda} + \frac{1}{h_o \eta \beta}}$$

定义 肋化系数: $\beta = A_o / A_i$ $Q \beta > 1, \therefore \eta \beta > 1$

$\therefore k$



§ 10-2 复合换热时的传热计算



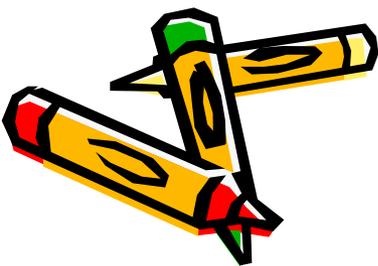
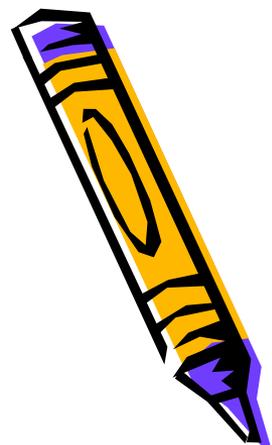
$$q_c = h_c (t_w - t_f) \quad (W / m^2)$$

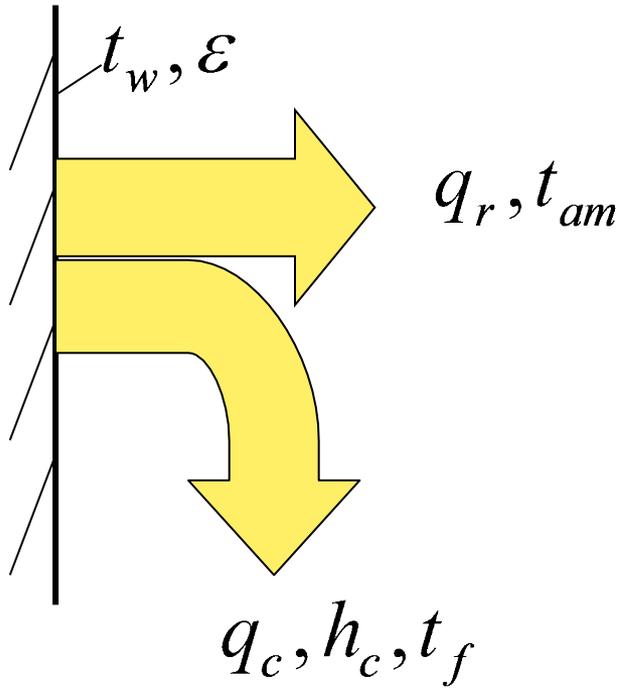
q_r, t_{am}

$$q_r = \varepsilon C_b \left[\left(\frac{T_w}{100} \right)^4 - \left(\frac{T_{am}}{100} \right)^4 \right] \quad (W / m^2)$$

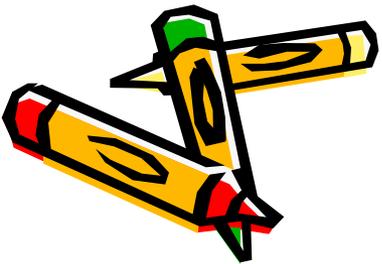
$$q_c, h_c, t_f = \left\{ \varepsilon C_b \frac{\left[\left(\frac{T_w}{100} \right)^4 - \left(\frac{T_{am}}{100} \right)^4 \right]}{t_w - t_f} \right\} (t_w - t_f)$$

$$= h_r (t_w - t_f)$$





$$q = q_c + q_r = (h_c + h_r)(t_w - t_f) \\ = h(t_w - t_f)$$



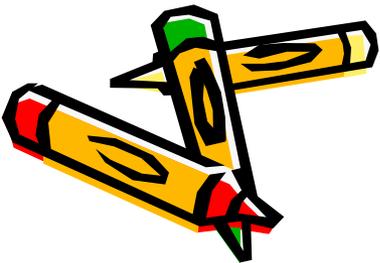
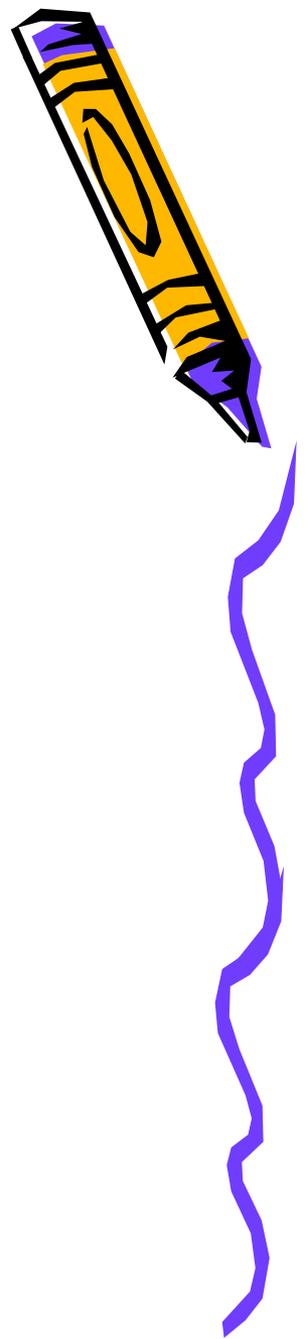
§ 10-3 换热器的型式和基本构造

一、分类

1. 按结构型式分：

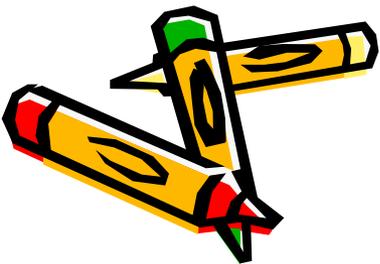
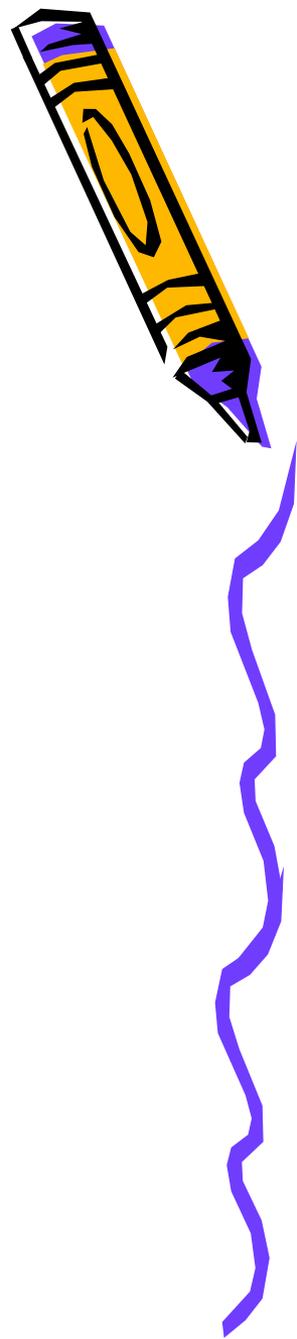
1) 间壁式：冷、热流体被固体壁面隔开。

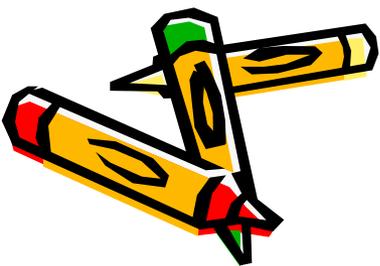
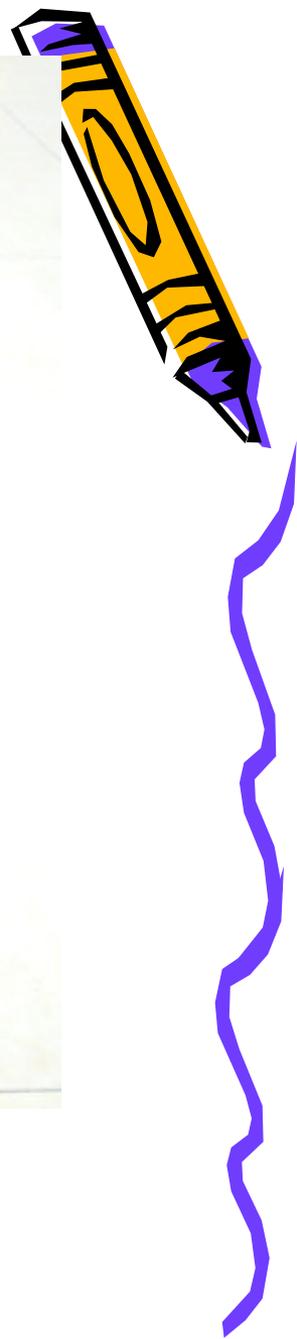
如：暖风机、冷凝器、蒸发器等。





暖风机

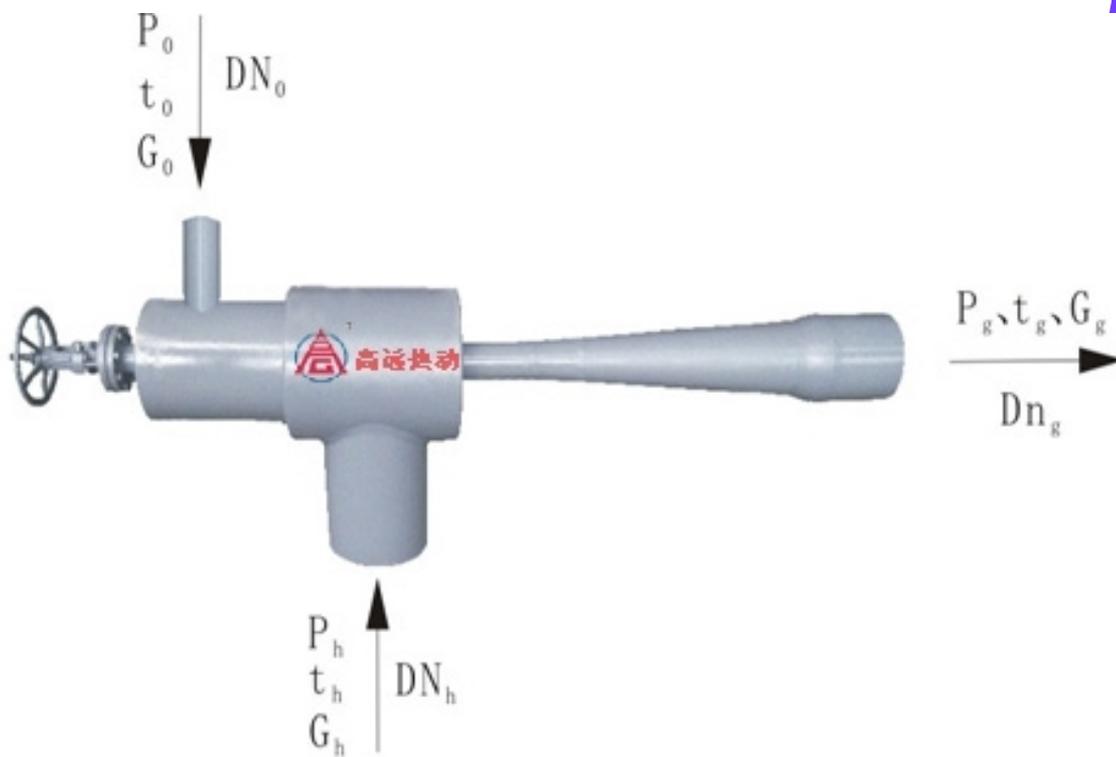




风冷冷凝器

2) 混合式：冷、热流体互相混合。

如：喷淋式冷却塔、蒸汽喷射器。



3) 蓄热式(回转式): 冷、热流体交替流过壁面, 壁面周期性地吸热和放热, 壁面除了换热, 还有蓄热作用。

如: 电站锅炉中的回转式空气预热器。



2. 按流动方式分：

1) 顺流：冷、热流体平行、同向流动。

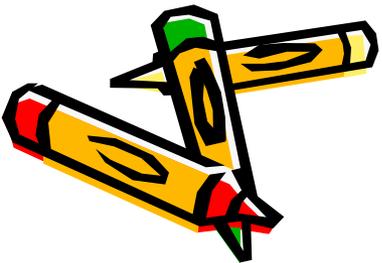
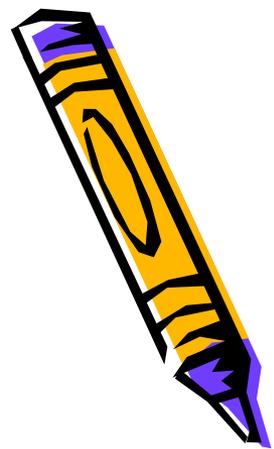
热流体 

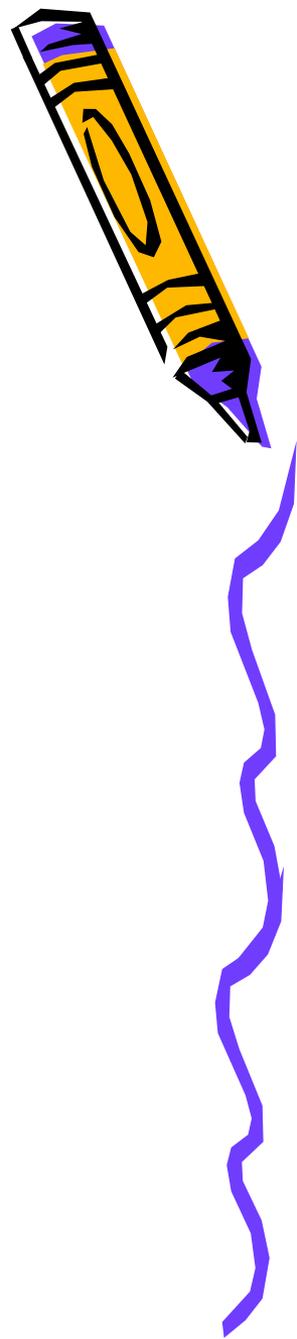
冷流体 

2) 逆流：冷、热流体平行、反向流动。

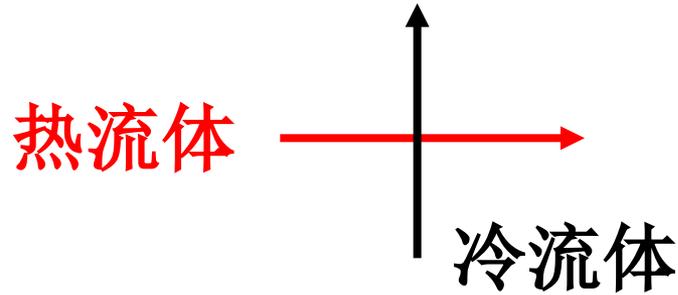
热流体 

冷流体 

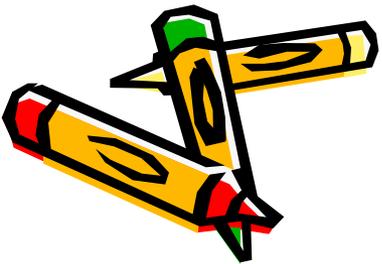
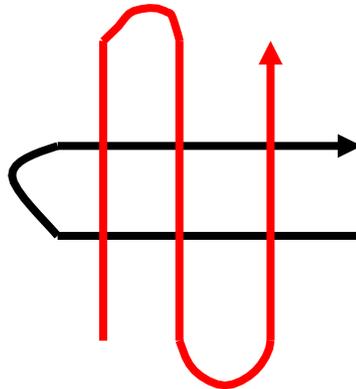




3) 交叉流：彼此垂直交叉流动。



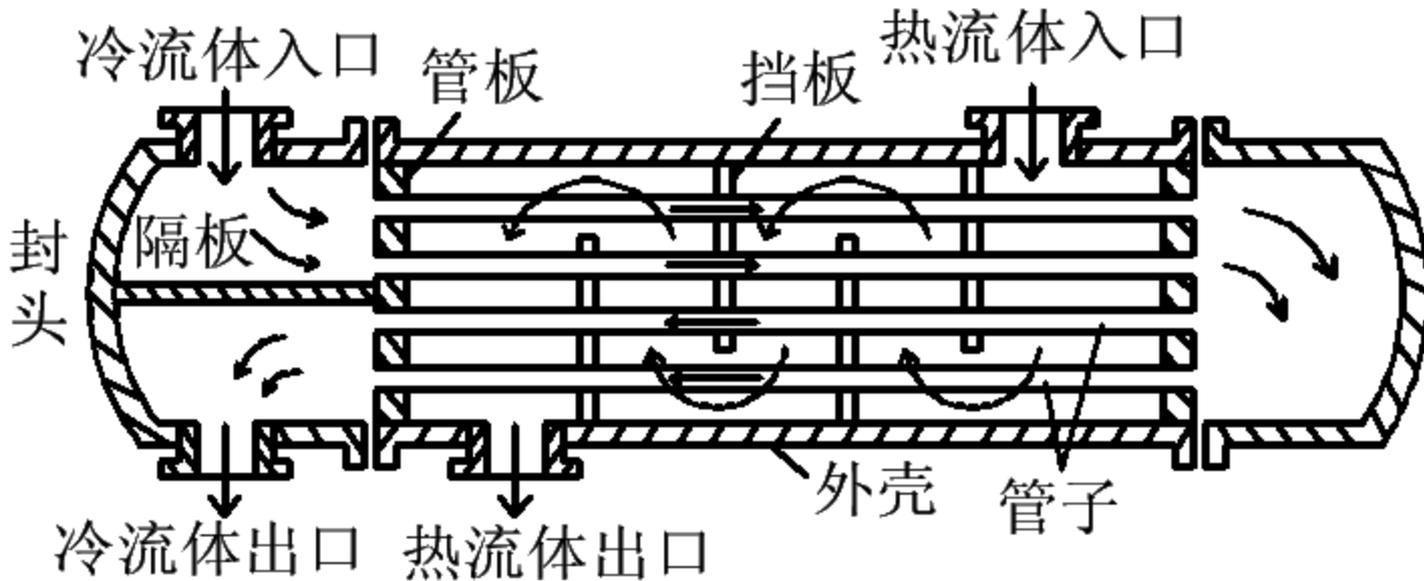
4) 混合流：多种流动的混合。

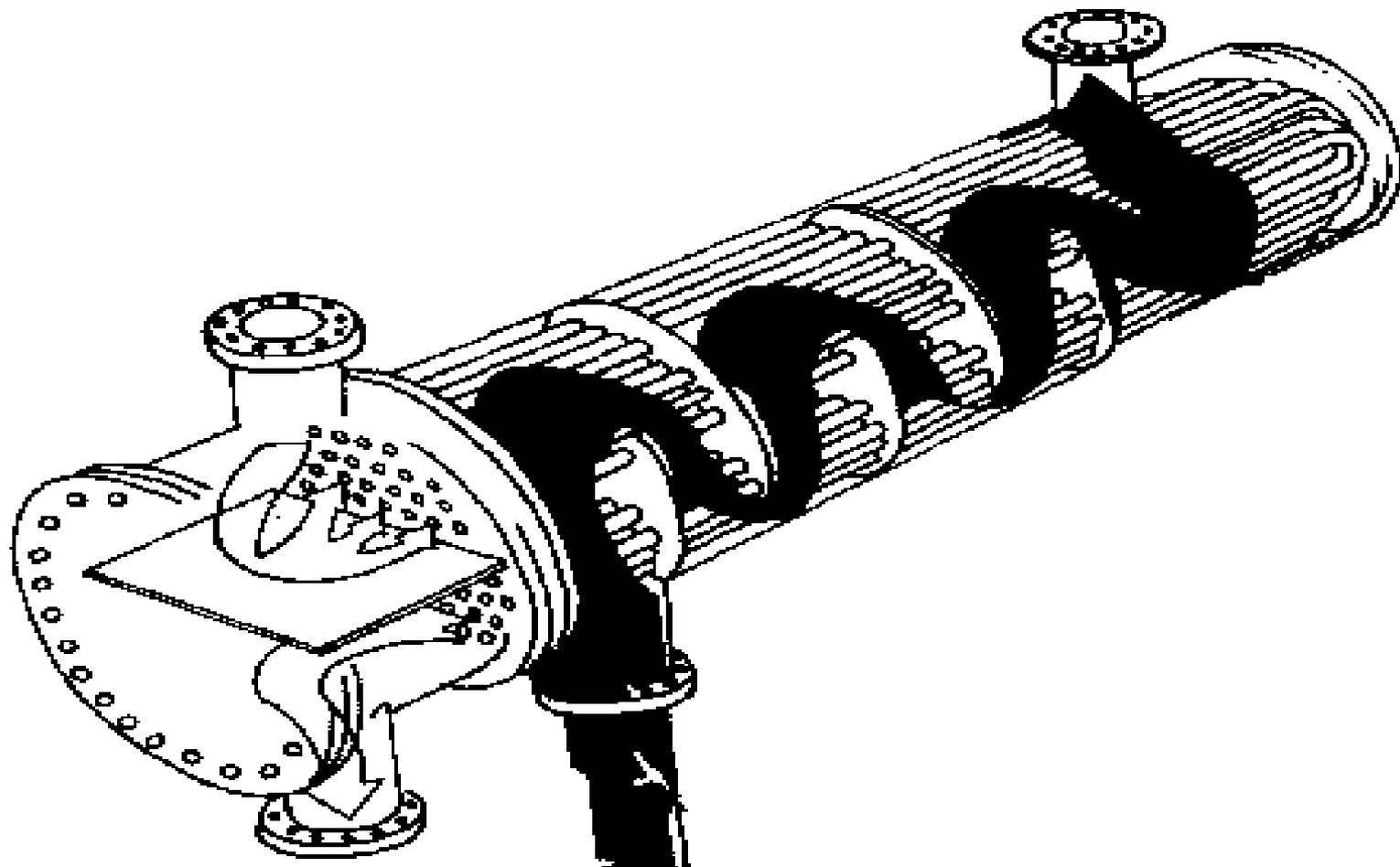


二、换热器的基本构造

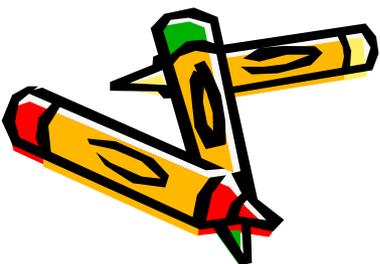
主要讨论间壁式换热器（应用最广）

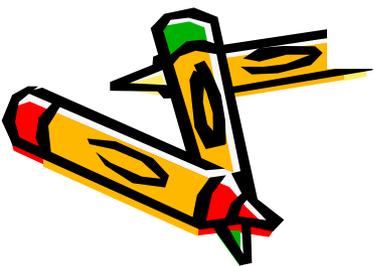
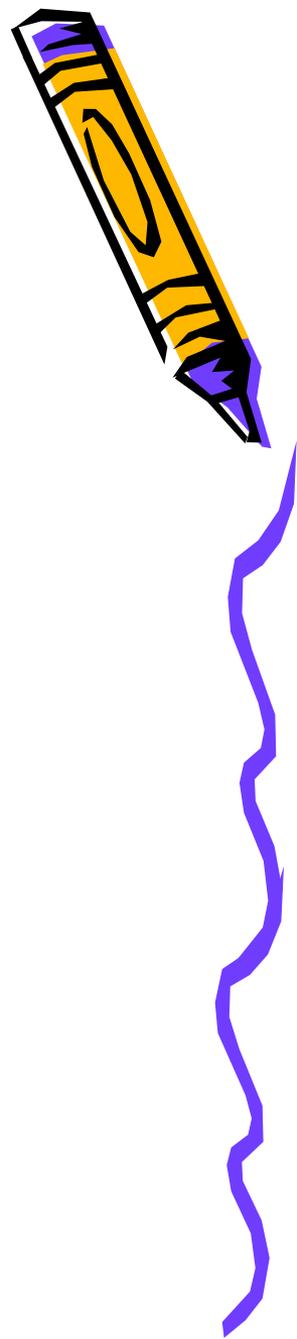
1. 管壳式换热器



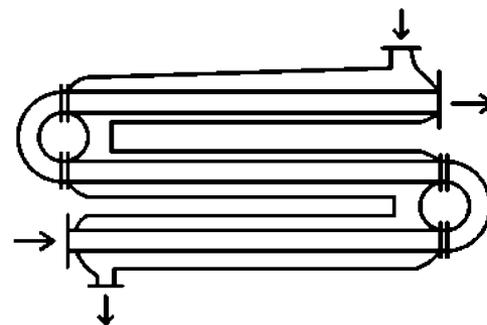
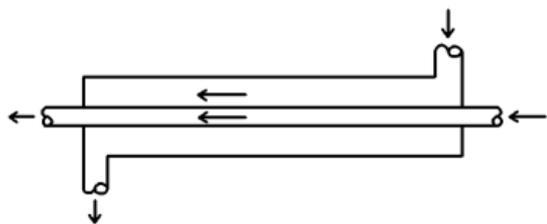


2-1型换热器

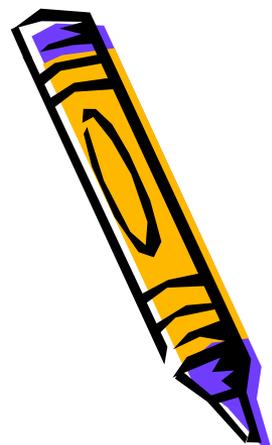
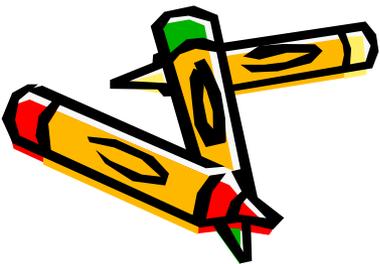


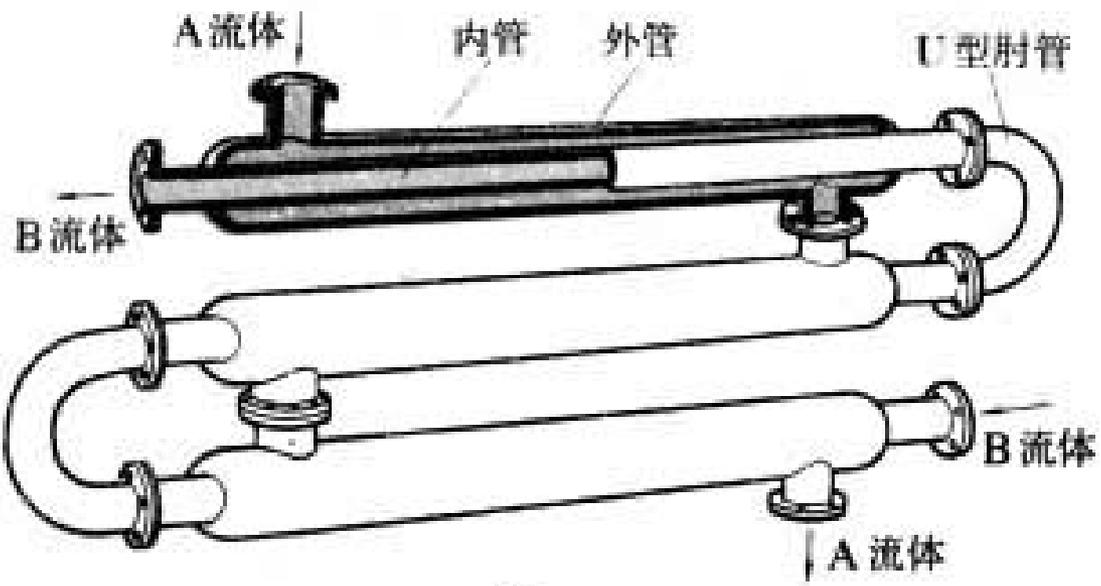


2. 套管式换热器

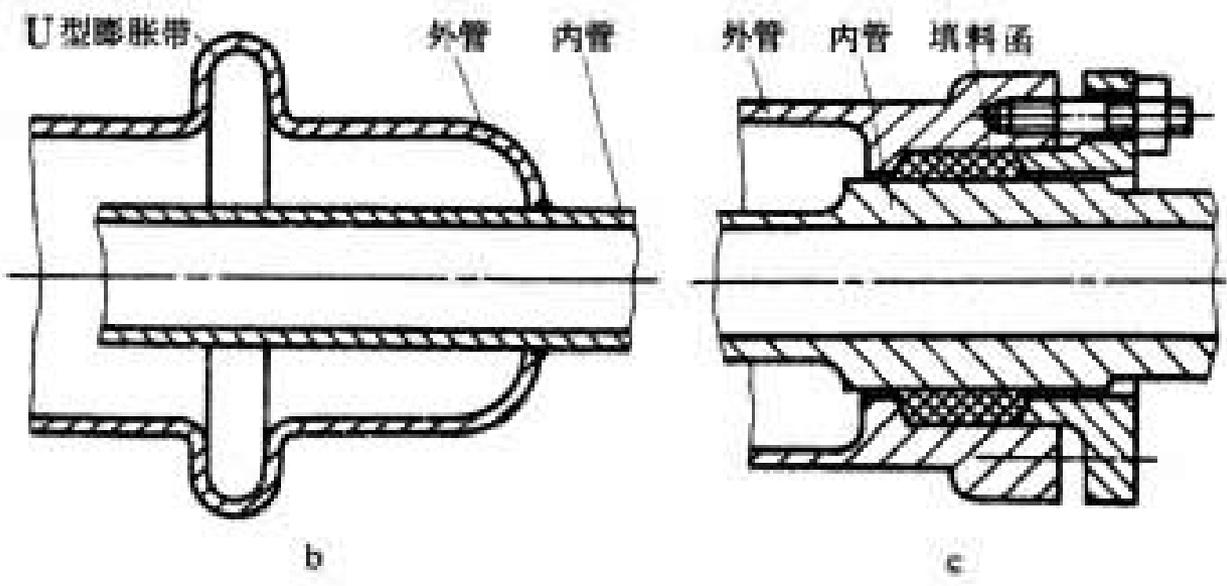


适用于传热量不大或流体流量不大的情形。





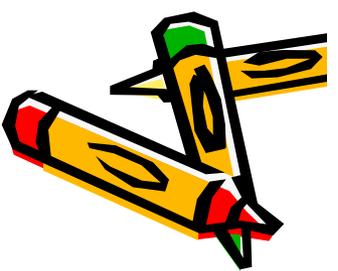
a



b

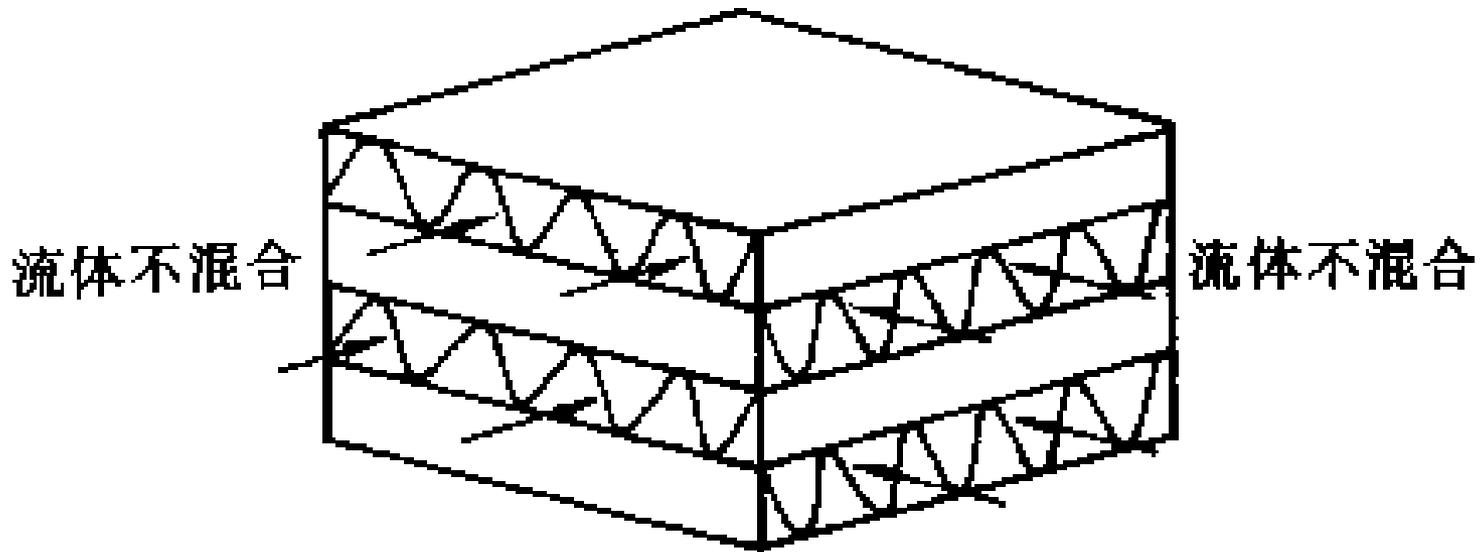
c

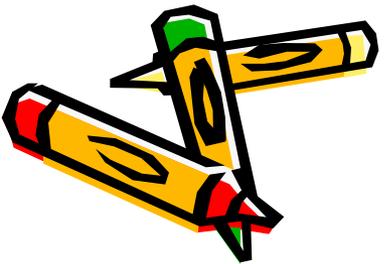
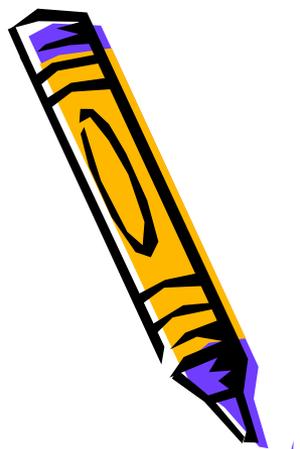
套管式换热器的结构



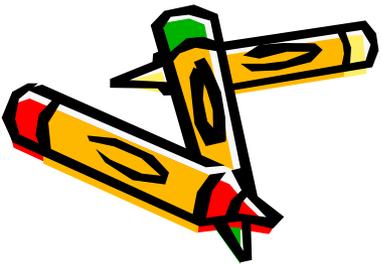
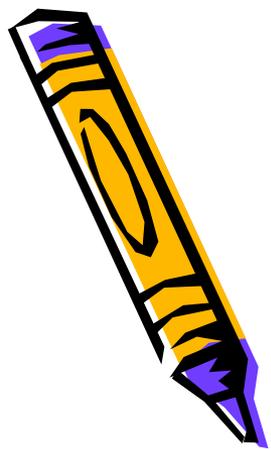
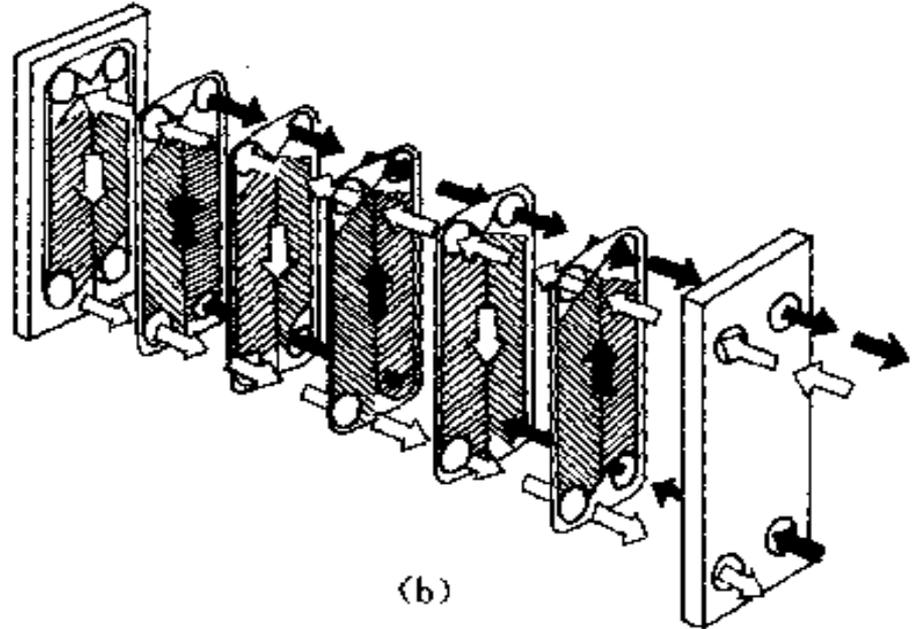
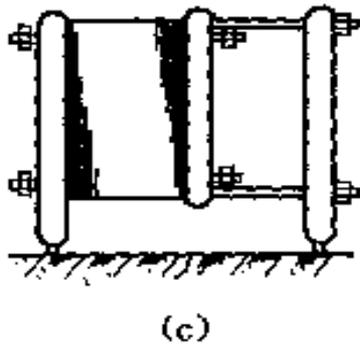
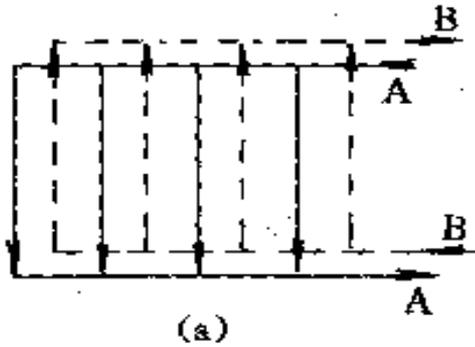
3. 交叉流换热器

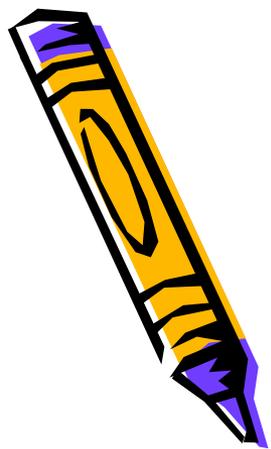
它是间壁式换热器的又一种主要型式。根据换热表面结构的不同又可有管束式、管翅式及板翅式等的区别。





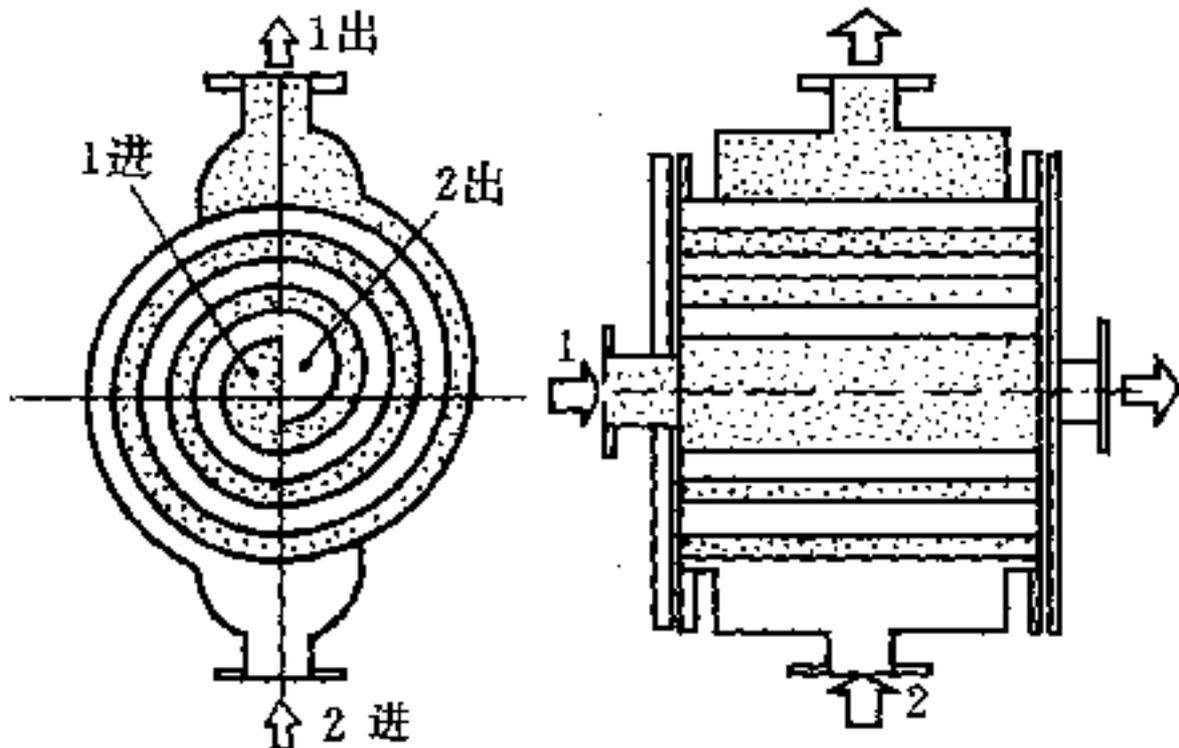
4. 板式换热器

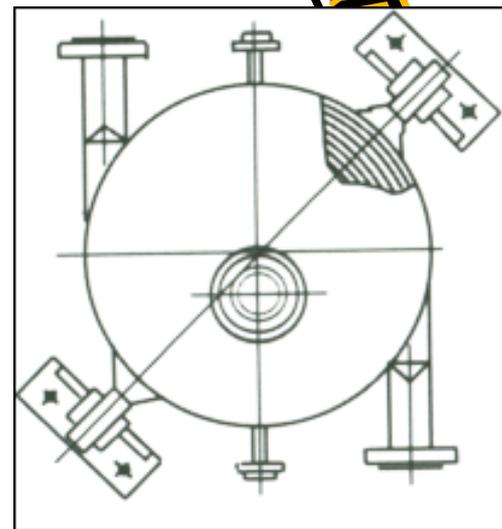
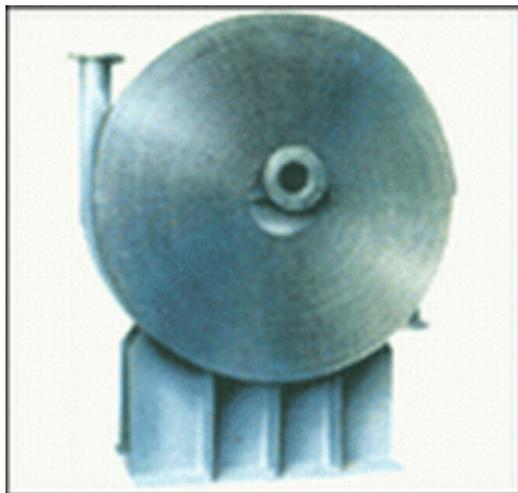




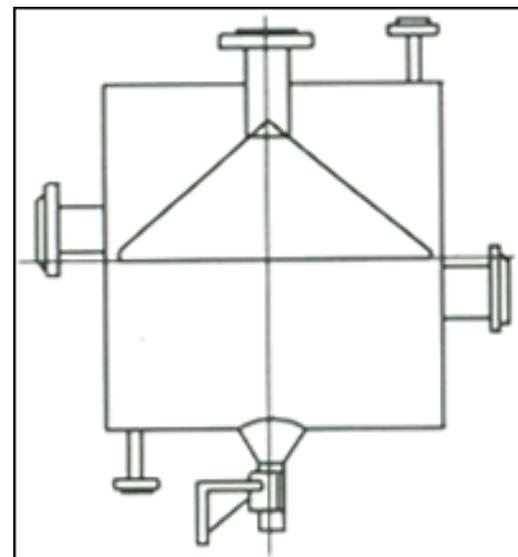
5. 螺旋板式换热器

这种换热器换热效果较好，缺点是换热器的密封比较困难。

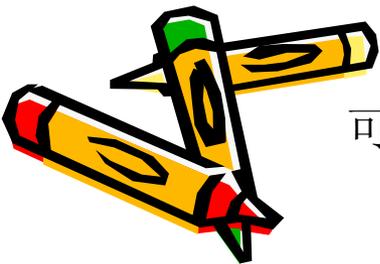




不可拆式 (I型)



可拆式 (II型)



§10-4 平均温差

传热方程的一般形式:

$$\Phi = KA \Delta t_m$$

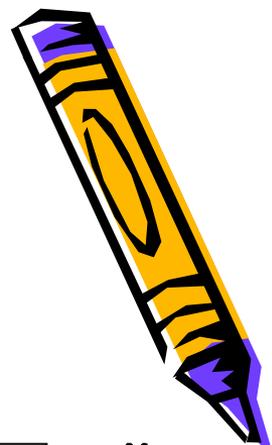
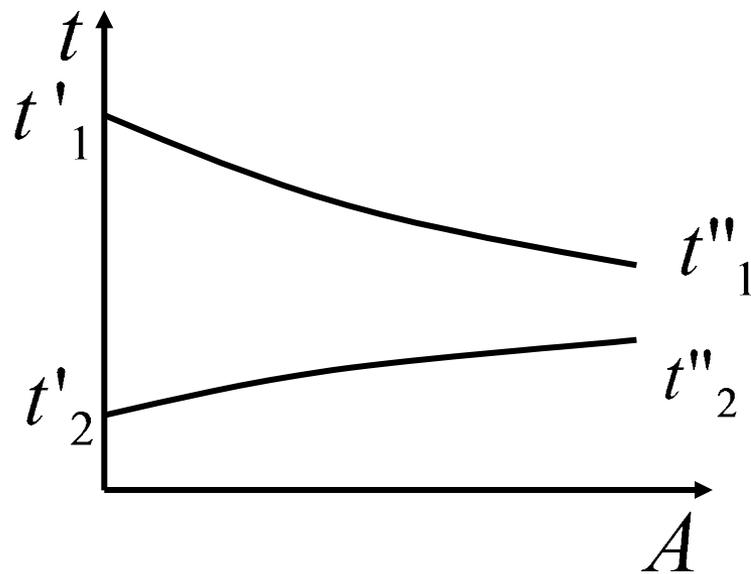
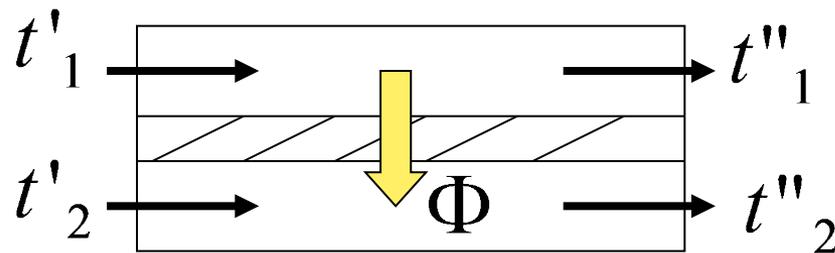
一. 顺流

入口用“'”,出口用“”

热流体:由 $t'_1 \rightarrow t''_1$

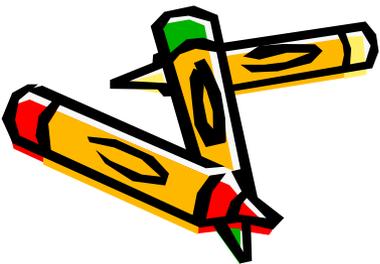
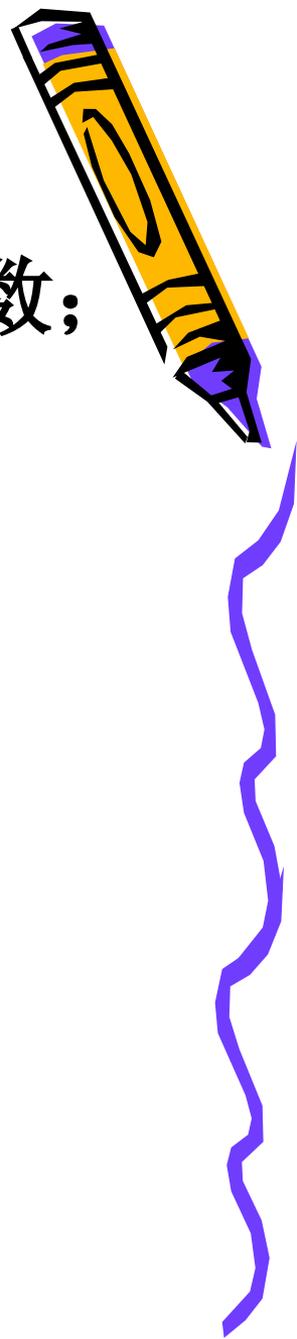
冷流体:由 $t'_2 \rightarrow t''_2$

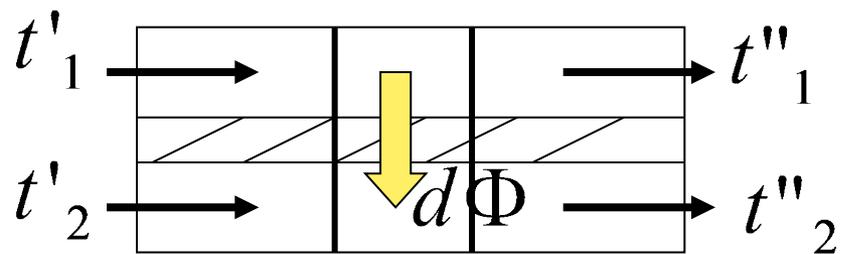
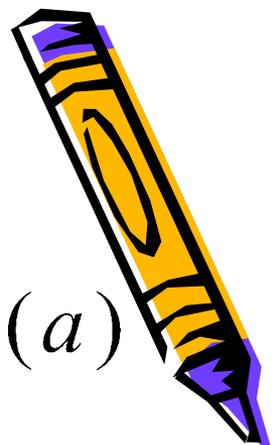
即冷、热流体的温差沿换热面是变化的.



假设:

- (1) 质量流量 M_1 、 M_2 ，比热容 c_1 、 c_2 为常数；
- (2) 传热系数 K 为常数；
- (3) 换热器无散热损失；
- (4) 换热面沿流动方向的导热热量可以忽略不计。

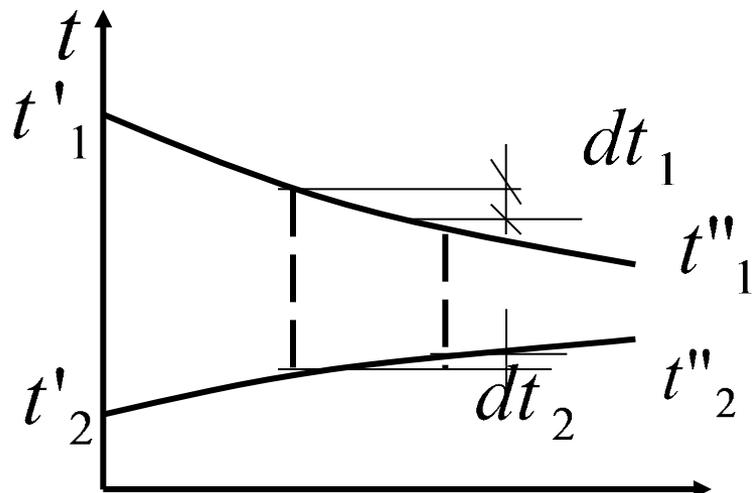




传热方程:

$$d\Phi = K \cdot \Delta t \cdot dA \quad (a)$$

热平衡方程:

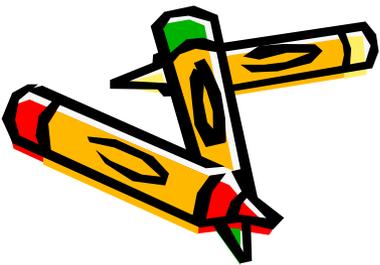


热流体 $d\Phi_1 = -M_1 c_1 dt_1 \quad (b)$

冷流体 $d\Phi_2 = M_2 c_2 dt_2 \quad (c)$

稳态、无散热 $d\Phi = d\Phi_1 = d\Phi_2$

冷、热流体温差 $\Delta t = t_1 - t_2 \quad (d)$

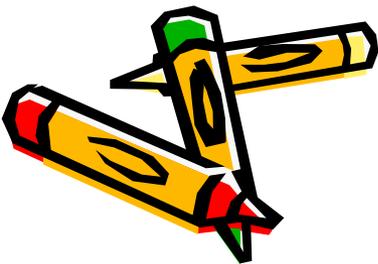
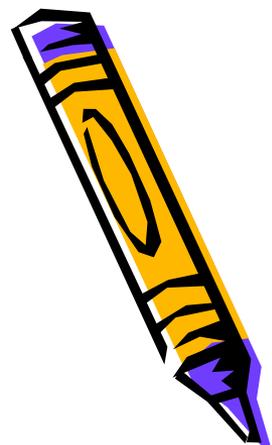


对式 (d) 微分, 得

$$\begin{aligned}d(\Delta t) &= dt_1 - dt_2 \\&= -\left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2}\right) d\Phi \\&= -\left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2}\right) K \cdot \Delta t \cdot dA\end{aligned}$$

分离变量, 并积分

$$\int_{\Delta t'}^{\Delta t_x} \frac{d(\Delta t)}{\Delta t} = - \int_0^{A_x} \left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2}\right) K \cdot dA$$



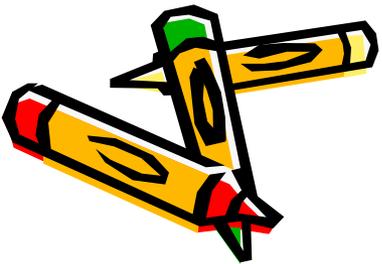
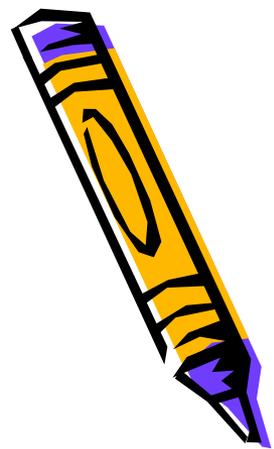
$$\ln \frac{\Delta t_x}{\Delta t'} = -K \cdot \left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2} \right) \cdot A_x$$

$$\therefore \Delta t_x = \Delta t' \cdot e^{-K \cdot \left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2} \right) \cdot A_x}$$

温差沿换热面按对数曲线变化

出口: $A_x = A$, $\Delta t_x = \Delta t''$

$$\therefore \frac{\Delta t''}{\Delta t'} = e^{-K \cdot \left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2} \right) \cdot A}$$

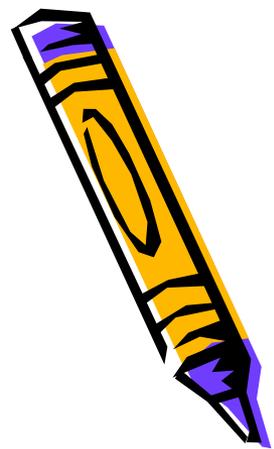
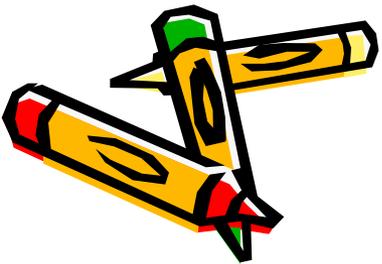


$$\Phi = KA \Delta t_m = \int_0^A K \Delta t_x dA$$

$$\therefore \Delta t_m = \frac{1}{A} \int_0^A \Delta t_x dA$$

$$= \frac{1}{A} \int_0^A \Delta t' \cdot e^{-K \cdot \left(\frac{1}{M_1 c_1} + \frac{1}{M_2 c_2} \right) \cdot A_x} dA$$

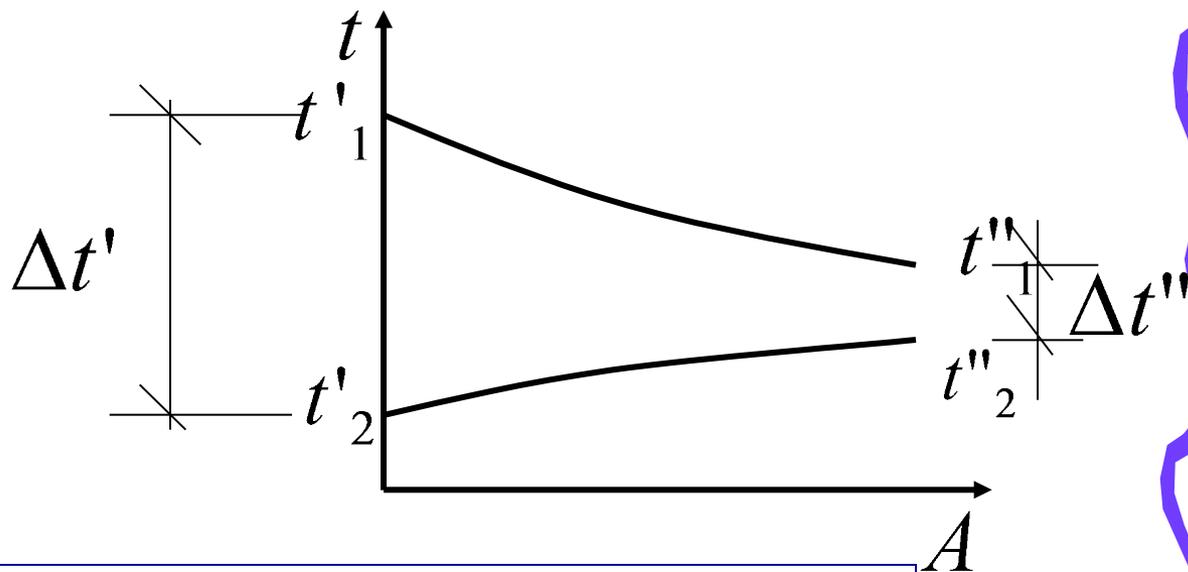
$$= \frac{\Delta t'' - \Delta t'}{\ln \frac{\Delta t''}{\Delta t'}}$$



$$\Delta t_m = \frac{\Delta t' - \Delta t''}{\ln \frac{\Delta t'}{\Delta t''}} = \frac{\text{端差大的} - \text{端差小的}}{\ln \frac{\text{大}}{\text{小}}}$$

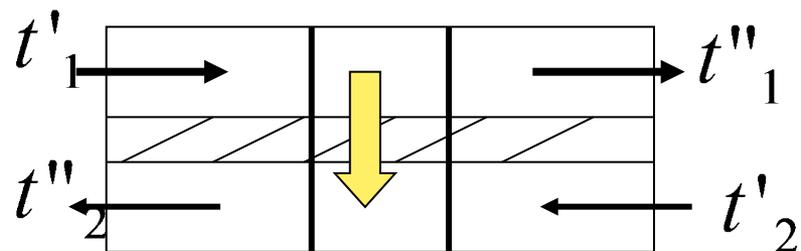
$$\Delta t' = t'_1 - t'_2$$

$$\Delta t'' = t''_1 - t''_2$$



端差是冷、热两种流体的温差

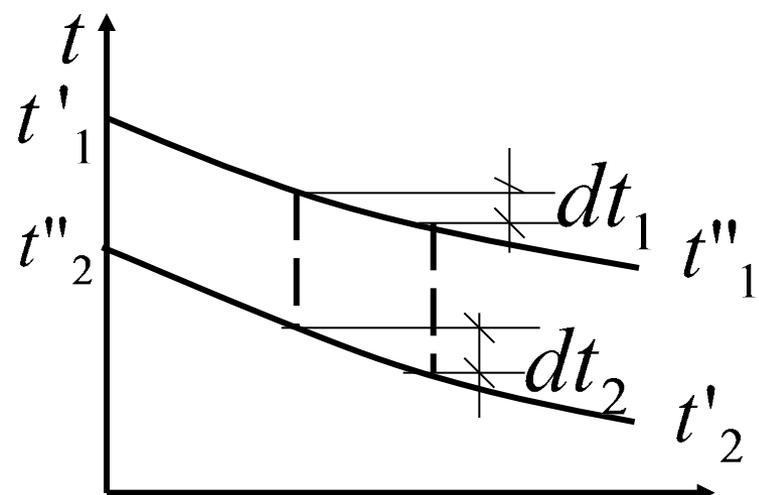
二. 逆流



$$d\Phi = K \cdot \Delta t \cdot dA \quad (a)$$

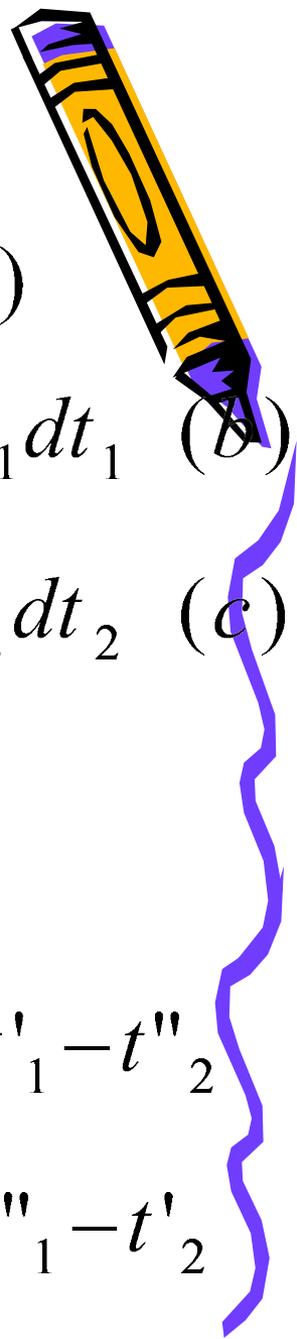
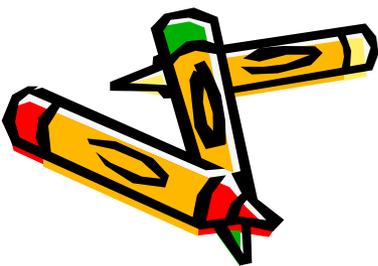
$$\text{热流体 } d\Phi_1 = -M_1 c_1 dt_1 \quad (b)$$

$$\text{冷流体 } d\Phi_2 = -M_2 c_2 dt_2 \quad (c)$$



$$\Delta t = t_1 - t_2 \quad (d)$$

$$\Delta t_m = \frac{\Delta t' - \Delta t''}{\ln \frac{\Delta t'}{\Delta t''}} \quad \begin{cases} \Delta t' = t'_1 - t''_2 \\ \Delta t'' = t''_1 - t'_2 \end{cases}$$



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